

**SAPTHAGIRI COLLEGE OF ENGINEERING**  
Department of Civil Engineering  
Internal Assessment Test -III

Subject: Strength of Materials  
Semester/Section: 5<sup>th</sup> SEM  
Duration: 90 Minutes

Sub Code: 17CV32

Name of the course instructor:

Max Marks: 30

PROF. GEETHA, I.S.

Date: 22.11.2018

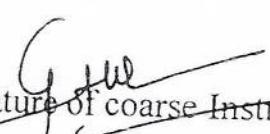
Time: 1.00PM to 2.30 PM

Note: Answer any two full questions, choosing one from each module

Question No.	Questions	Marks	BLT	CO's
<b>Module 4</b>				
1 a.	Differentiate between long column and short column.	5	L2	CO4
b.	Derive an expression of hoop stress and longitudinal stress of thin walled pressure vessels. <b>OR</b>	10	L2	CO4
2 a.	Differentiate between thin and thick cylinders.	5	L2	CO4
b.	Derive Lame's equation for radial and hoop stresses for thick cylinder subjected to internal and external fluid pressure.	10	L2	CO4
<b>Module 5</b>				
3 a.	Derive the Euler's equation for buckling load on an elastic column with both ends pinned.	7	L2	CO5
b.	A hollow rectangular cast iron column has external dimensions of 150mmX200mm and thickness of 25mm. The column is 5m long with both ends fixed. If E for column material is 120GPa, compute the critical load on this column by Euler's formula. Compare the value of load obtained by Rankine's formula. Take yield stress 300MPa and $\alpha=1/1600$ . <b>OR</b>	8	L4	CO5
4 a.	Derive the Euler's equation for buckling load on an elastic column with both ends fixed.	7	L2	CO5
b.	A hollow cylindrical cast iron column is 4m long both ends fixed. Design the column to carry axial load of 250KN. Use Rankine's formula. The internal diameter may be taken as 0.8 times the external diameter. Take E=550N/mm <sup>2</sup> , $\alpha=1/1600$	8	L4	CO5

Students should be able to understand the analysis of thin and thick cylinders.

Students should be able to understand the basic concept of analysis and design of structural elements such as columns and struts.

  
Signature of course Instructor

  
Principal  
Sapthagiri College of Engineering  
Chikkasandra, Hesaraghatta Road,  
Bangalore-560 057

Scrutinized by

1)   
2) 

HOD:



SAPTHAGIRI COLLEGE OF ENGINEERING, BANGALORE-57

DEPARTMENT OF CIVIL ENGINEERING

SCHEME & SOLUTIONS

Semester: III

Subject : Strength of Materials

Duration: 90 minutes

Staff Name: Geetha.T.S

Date: 22.7.2018

Subject code: 17CV32

Marks: 30

Signature: GTS

INTERNAL TEST

Qs No	Solutions	Marks Allocated
1.a	<p><u>Short column</u></p> <p><math>\frac{d_{eff}}{b} \leq 12</math></p> <p>Buckling tendency low</p> <p>Failure by crushing</p> <p>Load carrying capacity high</p>	<p><u>Long column</u></p> <p><math>\frac{d_{eff}}{b} &gt; 12</math></p> <p>Buckling tendency high</p> <p>Failure by buckling</p> <p>Load carrying capacity low</p>
b	<p>Fig</p> <p><math>P = P.d.L</math></p> <p><math>T = \sigma_c \times 2(\pi t)</math></p> <p><math>\sigma_c = \frac{Pd}{2t}</math></p>	<p><math>P_L = P \cdot \pi d^2 / 4</math></p> <p><math>P_R = \sigma_s \pi \cdot d \cdot t</math></p> <p><math>\sigma_s = \frac{Pd}{4t}</math></p> <p><math>\boxed{\sigma_c = 2\sigma_s}</math></p>
2.a	<p>- <math>t &lt; \frac{1}{10} r_i</math></p> <p>- radial shear stress neglected</p> <p>- hoop stress uniformly distributed</p> <p>- tires, gas storage tanks</p>	<p>- <math>t \geq \frac{1}{10} r_i</math></p> <p>- radial shear stress considered</p> <p>- varies parabolically</p> <p>- Gun Barrels</p>

SAPTHAGIRI COLLEGE OF ENGINEERING, BANAGLORE - 57  
 DEPARTMENT OF CIVIL ENGINEERING  
 SCHEME AND SOLUTIONS

SEMESTER:  
 SUBJECT:  
 DURATION:  
 STAFF NAME:

DATE:  
 SUBJECT CODE:  
 MARKS:  
 SIGNATURE:

INTERNAL TEST

Qs. No.	Solutions	Marks Allocated
b.	$I = \pi l^3 / 32 \times 10^6$	
	$P = \frac{4\pi^2 EI}{l^2} = 13.6 \times 10^6 \text{ N (Euler's) } - 4\text{m}$	8M
	$P = \frac{\alpha A}{1 + \alpha (\frac{l}{R})^2} = 1.75 \times 10^6 \text{ N (Rankine) } - 4\text{m}$	
	$P_{\text{Euler's}} > P_{\text{Rankine}}$	
4a	$BM, M = - (Py)$	
	Net Moment = $M + Mo$	
	$M + Mo = EI \frac{d^2y}{dx^2}$	
	$-Py + Mo = EI \frac{d^2y}{dx^2}$	
	$\frac{d^2y}{dx^2} + \frac{Py}{EI} = \frac{Mo}{EI}$	
	$y = C_1 \cos \sqrt{\frac{P}{EI}} x + C_2 \sin \sqrt{\frac{P}{EI}} x + \frac{Mo}{P}$	
	When $x=0, y=0$ -	$C_1 = -\frac{Mo}{P}$
	$\frac{dy}{dx} = 0$	$C_2 = 0$

**SAPTHAGIRI COLLEGE OF ENGINEERING**  
**Department of Engineering Mathematics**  
**Internal Assessment -III**

Subject: Engineering Mathematics - III

Sub Code: 17MAT31

Semester/Section: III SEM

Max Marks: 30

Duration: 1.5 hours

Date: 22-11-2018

Note: Answer any two full questions, choosing one from each module

Question No.	Questions	Marks	BLT	CO's												
<b>Module-1</b>																
1 a.	Using Newton-Raphson method find the real root of $x \log_{10} x = 1.2$ correct to three decimal places.  Find $f(1.4)$ from the following table	5	L1 & L2	C05												
	<table border="1" style="display: inline-table; vertical-align: middle;"> <tr><td>x</td><td>1</td><td>2</td><td>3</td><td>4</td><td>5</td></tr> <tr><td>F(x)</td><td>10</td><td>26</td><td>58</td><td>112</td><td>194</td></tr> </table>	x	1	2	3	4	5	F(x)	10	26	58	112	194	5	L1 & L2	C05
x	1	2	3	4	5											
F(x)	10	26	58	112	194											
c.	Find the equation of the cubic curve passing through the points $(4, -43), (7, 83), (9, 327), (12, 1053)$ using Newton divided difference formula and find $F(10)$ .	5	L1 & L2	C05												
OR																
2 a.	Find the root of the equation $\cos(x) = xe^x$ using Regula-Falsi method correct to four decimal places, in the interval $(0, 1)$ .	5	L1 & L2	C05												
b.	Using Lagrange's interpolation formula find a polynomial which passes through the points $(0, -1), (1, 0), (3, 6), (4, 12)$ .	5	L1 & L2	C05												
c.	Using Simpson $(3/8)^{th}$ rule evaluate $\int_0^{3\pi/4} e^{\sin x} dx$ by dividing the interval into six equal parts.	5	L1 & L2	C05												
<b>Module-2</b>																
3 a.	If $\vec{F} = 3xy\hat{i} - y^2\hat{j}$ , evaluate $\int_C \vec{F} \cdot d\vec{r}$ where $C$ is arc of parabola $y = 2x^2$ from $(0,0)$ to $(1, 2)$ .	5	L2	C06												
b.	Apply Green's theorem to evaluate $\int_C (2x^2 - y^2)dx + (x^2 + y^2)dy$ , where $C$ is the boundary of the $x$ -axis and the upper half of the circle $x^2 + y^2 = a^2$ .	5	L2	C06												
c.	Verify Stokes theorem for $\vec{F} = (x^2 + y^2)\hat{i} - 2xy\hat{j}$ taken around the rectangle by the lines $x = \pm a$ , $y = 0$ and $y = b$ .	5	L2	C06												
OR																
4 a.	Find the work done in moving a particle in the force field $\vec{F} = 3x^2\hat{i} + (2xz - y)\hat{j} + zk\hat{k}$ along the straight line from $(0,0,0)$ to $(2,1,3)$ .	5	L2	C06												

**DEPARTMENT OF MATHEMATICS**

**SCHEME & SOLUTION**

Sem. & Sec: III-Sem

Academic Year:

Date: 22/11/2018

Subject: Engineering Mathematics-II

Subject Code: 17MAT31

Duration: 1.5 hrs

Max. Marks: 30

**INTERNAL ASSESSMENT TEST - III**

Solutions

Marks  
Allocated

a)  $f(x) = x \log x - 1.2$ . Initial app<sup>n</sup>  $x_0 = 2$ .

$$f'(x) = 0.4342 + \log_{10} x$$

formula  $x_{n+1} = x_n - \frac{f(x_n)}{f'(x_n)}$

①

$$x_1 = x_0 - \frac{f(x_0)}{f'(x_0)} = 2 + \frac{0.5979}{0.7252} = 2.8132 \quad \text{--- } ①$$

$$x_2 = 2.7612, x_3 = 2.76064, x_4 = 2.7606 \quad \text{--- } ③ \quad [5]$$

b) Newton forward interpolation formula.

Table I<sub>st</sub> Row Value.

$x$	$f(x)$	$\Delta y$	$\Delta^2 y$	$\Delta^3 y$	$\Delta^4 y$
10	16		16	6	0
11					

$$f(x) = y_0 + P \Delta y_0 + \frac{P(P-1)}{2!} \Delta^2 y_0 + \frac{P(P-1)(P-2)}{3!} \Delta^3 y_0 \quad \text{--- } ①$$

$$P = \frac{14-1}{1} = 0.4$$

$$\text{Substitution } \therefore f(1.4) = 14.864 \quad \text{--- } ② \quad [5]$$

x) Table I<sub>st</sub> Row Value.

$x$	$f(x)$	$\Delta f(x)$	$\Delta^2 f(x)$	$\Delta^3 f(x)$
11	-43	42	16	1

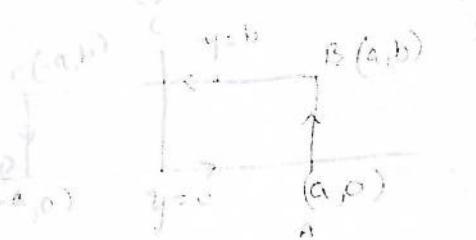
formulae. ①.

$$f(x) = f(x_0) + (x-x_0) \Delta f(x_0) + (x-x_0)(x-x_1) \Delta^2 f(x_0)$$

$$f(x) = x^3 - 4x^2 - 7x - 15 \quad \text{--- } ②$$

$$f(10) = 515$$

$$\oint \vec{F} \cdot d\vec{s} = \iint (\nabla \times \vec{F}) \cdot \hat{n} ds \quad \text{--- (1)}$$



$$\begin{aligned} \text{L.H.S. } \oint &= \int_C -ABx \, dA + \int_{AB} AB \, dA + \int_{BC} BC \, dA + \int_{CD} CD \, dA \\ &= -ab^2 - \frac{ab^3}{3} - ab^2 - ab^2 + \frac{ab^3}{3} \\ &= -4ab^2 \end{aligned} \quad \text{--- (2)}$$

$$\nabla \times \vec{F} = -4y \hat{k} \quad \text{R.H.S.} = \int_{-a}^a \int_{y=0}^b (-4y \hat{k}) \cdot \hat{k} dy dx = -b \int_{-a}^a dx = -4ab^2 \quad \text{--- (3)} \quad \boxed{5}$$

$$\text{Ans. Eq. } \frac{\partial C}{2} = \frac{y}{1} = \frac{x^2}{3} \quad \text{--- (4)}$$

Writing in parametric form or in terms of one variable.

$$y = \frac{z}{2}, \quad z = \frac{3x}{2}, \quad dz = \frac{3dx}{2} \quad \int 3x^2 dz + (2xz - y) dy + z dz \quad \text{--- (5)}$$

$$\int_{-2}^2 \left[ 3x^2 dz + \left( 2x\left(\frac{3x}{2}\right) - \frac{x}{2} \right) dy + \frac{3x}{2} \left( \frac{3dx}{2} \right) \right] = 16 \quad \text{--- (6)} \quad \boxed{5}$$

$$Q = xy + y^2 \quad Q = x^2 \quad \text{--- (7)} \\ \oint = \int_{C_1} + \int_{C_2} = \int_{C_1} (3x^3 + x^4) dx + \int_{C_2} 3x^2 dx \quad \text{--- (8)} \quad \text{--- (3)}$$

$$\text{L.H.S.} = \frac{19}{20} - 1 = -\frac{1}{20}$$

$$\text{L.H.S.} \iint_R \left( \frac{\partial Q}{\partial x} - \frac{\partial P}{\partial y} \right) dx dy = \iint_R (x - 2y) dy dx \quad \text{--- (9)} \quad \text{--- (10)} \quad \checkmark \quad \boxed{5}$$

$$= -\frac{1}{20}$$

USN 1 S G I M E

17ME34

SAPTHAGIRI COLLEGE OF ENGINEERING  
DEPARTMENT OF MECHANICAL ENGINEERING  
FIRST INTERNAL ASSESSMENT - QDD SEMESTER 2018-19

## MECHANICS OF MATERIALS

Sem & Sec: 3<sup>rd</sup> 'A' & 'B'

Date: 21-09-2018

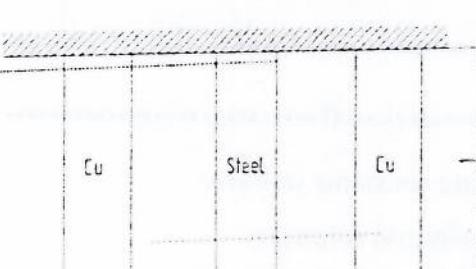
Time: 90 Min

Max. Marks: 30

Note: Answer any one full question from each Module

Q. NO.	MODULE - I	MARKS	RBT LEVEL	CO's
1. a.	Define (i) Hooke's Law (ii) Poisson's Ratio (iii) Bulk Modulus (iv) Rigidity Modulus	-04-	11	CO1
b.	Derive an expression for elongation of tapered bar with rectangular cross-section	-11-	11, 12, 13	CO1

OR

2. a.	Derive an expression for establishing the relationship between young's modulus and rigidity modulus.	-08-	11, 12, 13	CO1
b.	A composite bar consisting of three rods, each of 1m length and 500 mm <sup>2</sup> in cross-section as shown in fig. If the temperature rise in 40°C, estimate the load carried by each rod and elongation. Take E <sub>s</sub> = 200 GN/m <sup>2</sup> , E <sub>c</sub> = 100 GN/m <sup>2</sup> , α = 1.2 × 10 <sup>-5</sup> /°C, α <sub>c</sub> = 1.8 × 10 <sup>-5</sup> /°C.   All Dimensions are in meter	-07-	11, 12, 13	CO1

Q. NO.	MODULE - II	MARKS	RBT LEVEL	CO's
3. a.	For the round bar shown in fig., determine the magnitude of force P such that net deformation is 1mm. Take E <sub>s</sub> = 200 GPa & E <sub>a</sub> = 70 GPa.	-07-	11, 12, 13	CO1

Sapthagiri College of Engineering, Department of Mechanical Engineering, Bangalore-560057

Scheme and Solution

Sub Name:

Code: 14ME34 Class and Section:

Date:

Mechanics of Materials

III A E B

21/09/18

2018

Question Number	Solution	Marks Allocated
	MODULE 1	
1. a.	Each Definition → 1M	1M
b.	Sketch →	0M
-to	write width of element $b_e = b_1 - \frac{(b_1 - b_2)x}{l}$	1M
-to	Area of the element $A_e = b_e \times l$	1M
-to	Elongation of the element $\delta_e = \frac{P \cdot dx}{(b_1 - b_2)JE}$	2M
-to	Total Elongation $\delta_t = \int_0^l \delta_e = \int_0^l \frac{P \cdot dx}{(b_1 - b_2)JE}$	0M
-to	$\delta_t = \frac{P \cdot l}{JE(b_1 - b_2)} \log\left(\frac{b_1}{b_2}\right)$	3M
	BR	0M

2. a.	Sketch →	0M
-to	write strain in diagonal $DB = \frac{B'E}{DB} \rightarrow$	1M
-to	$DB = AB \cdot \sqrt{2} \quad \left. \begin{array}{l} DB = \frac{BB'}{\sqrt{B}} \\ BB' = \frac{BB}{\sqrt{B}} \end{array} \right\} \rightarrow$	1M
-to	strain in diagonal $DB = \frac{BB}{\sqrt{B}} \cdot \frac{1}{2} AB \sqrt{2}$	1M
-to	strain in diagonal $DB = \frac{1}{2} AB \sqrt{2} \rightarrow$	1M
-to	strain in diagonal $DB = \frac{C}{E} \left(1 + \frac{1}{m}\right) \rightarrow$	0M
	$E = \frac{C}{m} \left(1 + \frac{1}{m}\right) \rightarrow$	0M
		0.8M

Principal  
Sapthagiri College of Engineering  
Chikkasandra, Hesaraghatta Road,  
Bangalore-560 057

Scheme and Solution

Sub Name:

Code: IITME34 Class and Section:

Date:

MOM

III-A, B

21/09/18

Question Number	Solution	Marks Allocated
3b.	$\frac{T_s}{R} = \frac{T}{\alpha}$ $\rightarrow$ To find $dT = \frac{\partial T}{R} T \cdot \alpha^3 \cdot d\alpha \rightarrow$ $T = \frac{T_s}{R} \times J$ $J = \frac{T_s}{\alpha} = \frac{G_E}{\alpha} \rightarrow$	1M 2M 0.8M
4-a	$\epsilon_x = \frac{\sigma_x}{E} = \frac{\tau_y}{mE} = \frac{\tau_z}{mE}$ $\epsilon_y = \frac{\tau_y}{E} = \frac{\tau_z}{mE} = \frac{\tau_x}{mE}$ $\epsilon_z = \frac{\tau_z}{E} = \frac{\tau_x}{mE} = \frac{\tau_y}{mE}$ $\epsilon_v = \frac{(\tau_x + \tau_y + \tau_z)}{E} \left(1 - \frac{2}{m}\right) \rightarrow$ $\sigma_x + \sigma_y + \sigma_z = 0$ $\epsilon_v = 0$	3M 3M 1M 1M 0.8M
5.	To find $T_{max} = 3819.70 N-m \rightarrow$ $T_{max} = 1.25 T_{mean}$ $T_{mean} = 6319.65 N-m \rightarrow$ $\frac{1}{J} \cdot \frac{G_E}{\alpha} = 0.014 m \rightarrow$ $\frac{J}{\alpha} = \frac{G_E}{\ell} \Rightarrow \theta = 0.0811 rad \rightarrow$ $\theta = 4.65$	2M 1M 1M 1M 0.7M